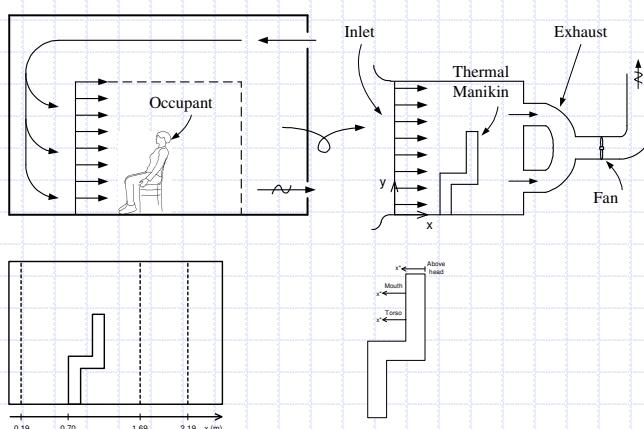


# CFD Manikin Benchmark Test - Mixing Ventilation

Claus Topp  
Lindab Ventilation A/S  
Denmark

Workshop on CFD Manikins  
ROOMVENT 2004, Coimbra, Portugal

## Experimental setup Mixing Ventilation



Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004



2

# Test Conditions

## CFD code

- ◆ Software: Fluent 6
- ◆ Turbulence model: Low-Re k- $\epsilon$  model
- ◆ Algorithm: SIMPLE
- ◆ Scheme: First order upwind

## Computer Simulated Person

- ◆ Heat flux: 38 W, convection
- ◆ No breathing
- ◆ Seated height: 1.38 m
- ◆ Surface area: 1.52 m<sup>2</sup>

## Grid

- ◆ CSP surface grid: 10,000 triangular elements
- ◆ Room grid: 292,421 tetrahedral cells
- ◆ Non-dimensional wall distance,  $y^+$ , to the centre of the first cell was below one across most of the surfaces

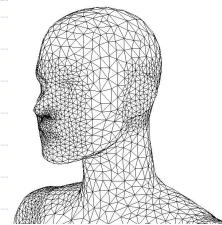
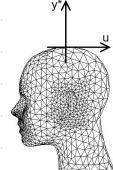
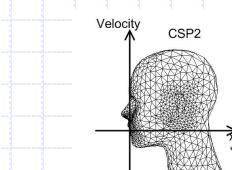
Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

3

# CFD manikin (CSP)

- ◆ Corresponds to manikin used in measurements
- ◆ CSP surface grid: 10,000 triangular elements
- ◆ Seated height: 1.38 m
- ◆ Surface area: 1.52 m<sup>2</sup>



Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

4

## Boundary conditions

	$U_o$ (m/s)	$T_o$ (°C)	$K_o$ (m <sup>2</sup> /s <sup>2</sup> )	$\epsilon_o$ (J/kg s)
Case 1	0.05	22	6.0E-04	4.8E-06
Case 2	0.20	22	9.6E-03	3.1E-04
Case 3	0.50	22	6.0E-02	4.8E-03

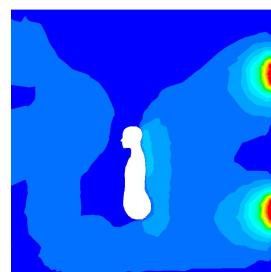
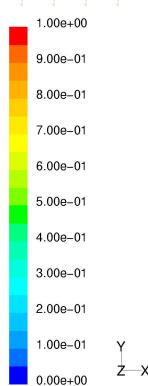
Floor: Surface concentration  $c_s$  of 200 g/m<sup>3</sup> with a density of  $\rho = 1.2$  kg/m<sup>3</sup>  
(different from benchmark case!)

Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

5

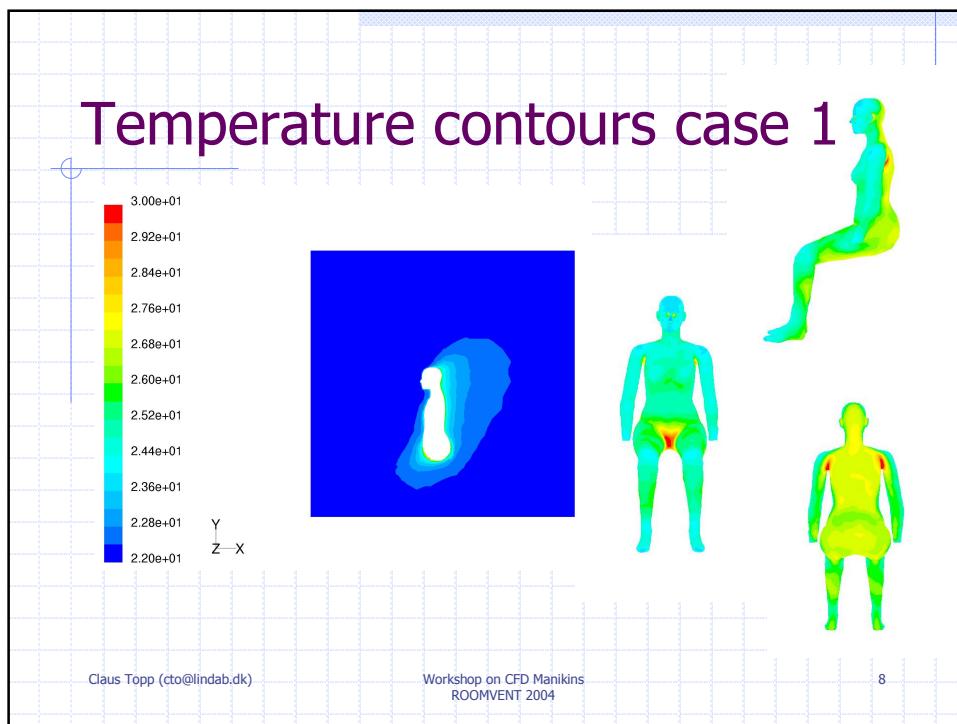
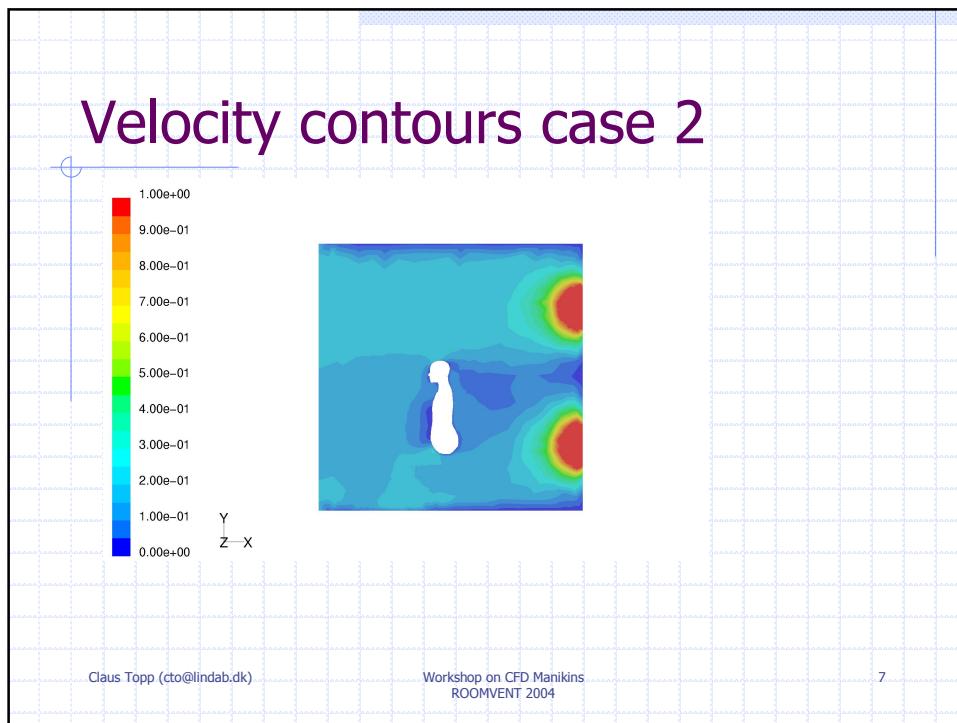
## Velocity contours case 1

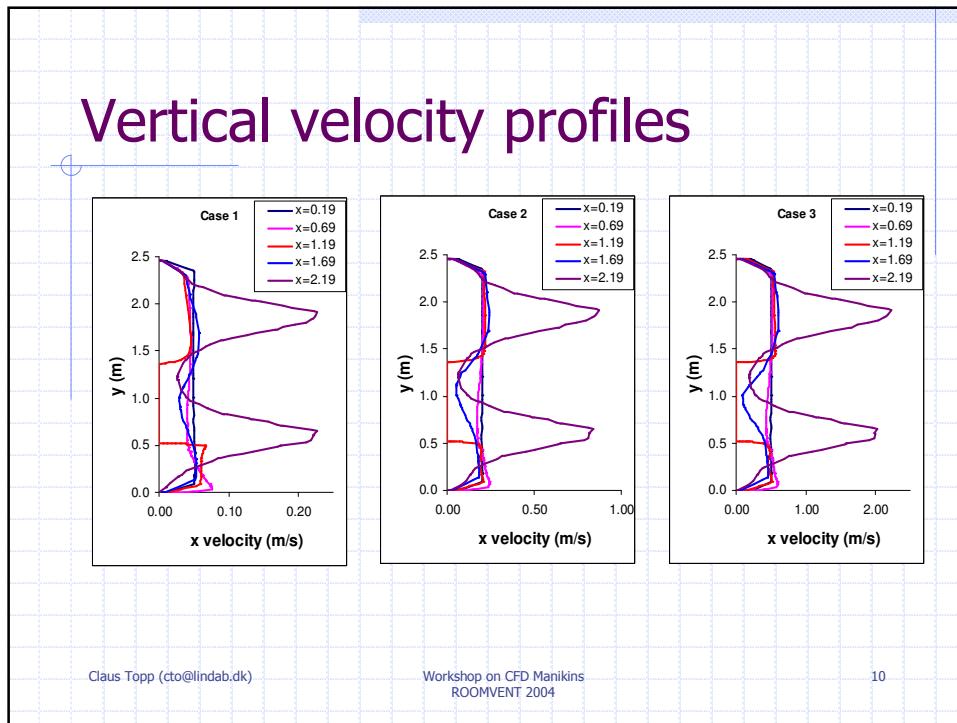
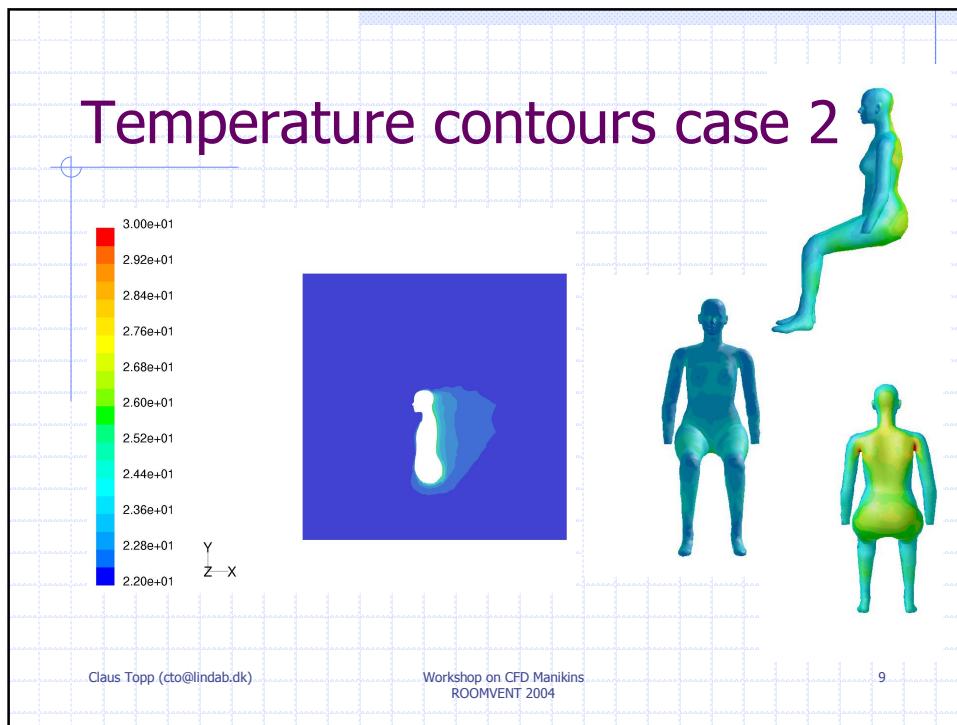


Claus Topp (cto@lindab.dk)

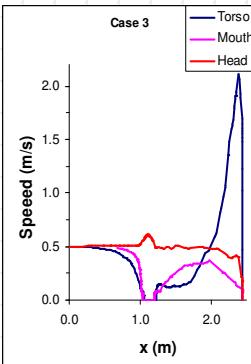
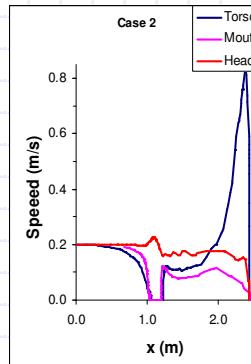
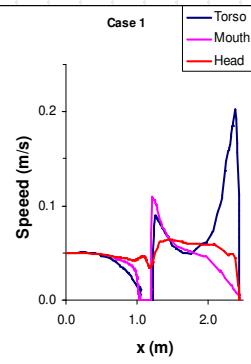
Workshop on CFD Manikins  
ROOMVENT 2004

6





## Horizontal velocity profiles

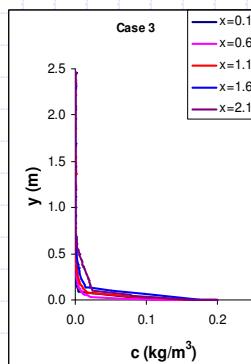
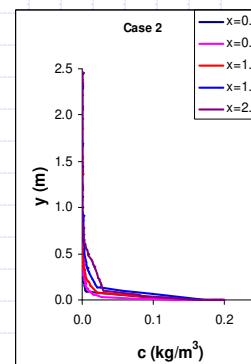
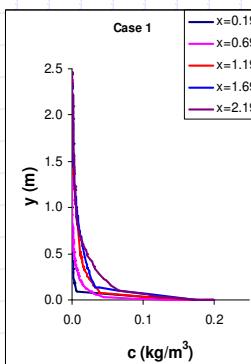


Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

11

## Concentration gradients

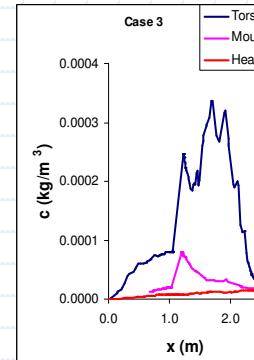
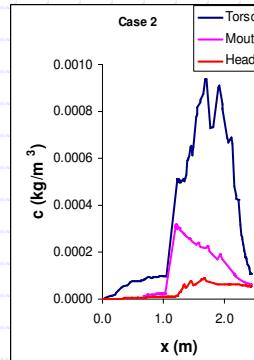
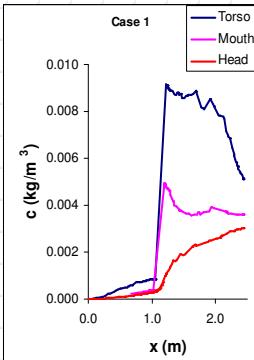


Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

12

## Concentration profiles

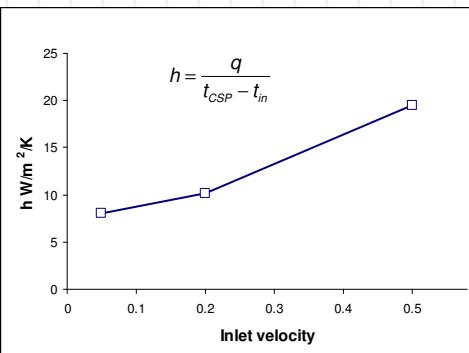


Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

13

## Convective heat transfer coefficient



$h$  = convective heat transfer coefficient ( $\text{W/m}^2\text{K}$ )  
 $q$  = heat flux ( $\text{W/m}^2$ )  
 $t_{CSP}$  = surface temperature of CSP ( $^\circ\text{C}$ )  
 $t_{in}$  = inlet temperature ( $^\circ\text{C}$ )

Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

14

# Papers

Topp, C., Hesselholt, P., Trier. M. R. and Nielsen, P. V.  
*Influence of Geometry of Thermal Manikins on Concentration Distribution and Personal Exposure*  
Proceedings of Healthy Buildings 2003, Singapore, 2003.

Topp, C., Hesselholt, P., Trier. M. R. and Nielsen, P. V.  
*Influence of Geometry of Thermal Manikins on Room Airflow*  
Proceedings of Healthy Buildings 2003, Singapore, 2003.

Topp, C.  
*Influence of Geometry of a Computer Simulated Person on Contaminant Distribution and Personal Exposure*  
Proceedings of ROOMVENT 2002, Copenhagen Denmark, 2002.

Topp, C., Nielsen, Peter V. and Sørensen, D. N.  
*Application of Computer Simulated Persons in Indoor Environmental Modeling*  
ASHRAE Transactions, Vol. 108 (2), p. 1084-1089, 2002.

Claus Topp (cto@lindab.dk)

Workshop on CFD Manikins  
ROOMVENT 2004

15